Please replace paragraph [00~~63~~]71 with the following amended paragraph: A.M. 1/2/07

[00~~63~~]71 Referring now to the drawings wherein like reference numerals are used to identify similar components in the various views, Figure 1 illustrates the principle of a belt drive used for an internal combustion engine in an axial view of the crankshaft. A continuous belt 11, such as a multi-groove V-belt or a toothed belt, moves across three inner belt pulleys: a belt pulley 12 which is fitted on the crankshaft and driven by same, and which drives the belt drive; a belt pulley 13, driven by the belt, for a compressor of an air conditioning system; and a belt pulley 14, driven by the belt 11, for an electric generator. The toothed belt rotates clockwise. Between the belt pulley 12 and the belt pulley 14, there extends a slack strand which is loaded by a tensioning roller 15 which is illustrated in different positions with different belt lengths. The tensioning roller 15 is arranged at a tensioning arm 19 so as to be pivotable around an axis of rotation A_2 which, at the same time, forms the central axis of a torsion spring assembly 20. The torsion spring assembly 20, at a first end, is secured in a rotationally ~~fast~~ fixed way to the engine block and, at is other end, carries the tensioning arm 19 with the tensioning roller 15 which is arrange radially relative to the central axis of the torsion spring assembly 20. The tensioning roller 15 is rotatable around an axis of rotation A_1 and is in contact with the smooth outer face of the belt. The tensioning arm 19 can be supported by a variable-length steering arm 16 on a rack. The steering arm 16 can comprise damping and/or spring elements. Of the various positions of the tensioning roller 15, the central position PN indicates the nominal position, whereas the adjoining positions PM2 and PE3 constitute tolerance regions, with the outer positions PM3 indicating the new condition prior to an initial rotation and position PE4 indicating an end stop beyond a permissible belt elongation and belt aging.

Please replace paragraph [00~~64~~]72 with the following amended paragraph: A.M. 1/2/07

[00~~64~~]72 Figure 2 shows the inventive belt tensioning device with its main parts with the cylindrical tensioning roller 15 positioned on the smooth outer face of the belt, a tensioning arm 19 and torsion spring assembly 20. The tensioning roller 15 is a lightweight plastic part which is rotatably supported on a bearing journal 21 at the tensioning arm 19. The tensioning arm 19 is a cranked carrier and comprises weight-reducing pockets 22 on one side. The tensioning arm 19 is held in a rotationally ~~fast~~ fixed way at the upper end of the torsion spring assembly, at its lower end, the torsion spring assembly 20 is secured by means (not shown in detail) in a rotationally ~~fast~~ fixed way to the engine block.

A.M. 1/2/07

Please replace paragraph [006~~6~~]74 with the following amended paragraph:

[00~~66~~]74 Figure 4 shows the complete assembly according to Figure 3B in a larger scale. It can be seen that the pockets 22 provided for weight-reduction are provided on one side of the tensioning arm 19. One or more of the pockets 22 could also be formed as through apertures. At the free end of the tensioning arm 19, there is shown the bearing journal 21 on which the tensioning roller 15 is rotatably supported by means of a roller bearing 23. The inner bearing race of the bearing is held by a screw 24 on the bearing journal 21. The tensioning roller 15 embraces the outer bearing race of the bearing in a form-fitting and positive way. The tensioning roller is thus rotatable around an axis of rotation A_1. At the other end of the tensioning arm 19, there is provided an eye 25 into which there is inserted a first bush 26 in a rotationally ~~fast~~ fixed way, such as by a press fit. Into the first bush 26, there is inserted one end of a bundle 27 of torsion springs 32, which spring bundle 27, in its entirety, is connected to the first bush 26 in a rotationally ~~fast~~ fixed way. The other end of the bundle 27 of torsion springs 32 is inserted into a second bush 28, which bundle 27, again, in its entirety, is connected to the second bush 28 in a rotationally ~~fast~~ fixed way. The above-described end of the tube 29 extends over the lower end of the first bush 26, with sleeves 30, 31 serving different functions being inserted between the tube 29 and the first bush 26. In the fitted condition, the lower end of the torsion spring assembly 20 is clamped into a holding mechanism in a rotationally ~~fast~~ fixed way, either directly via the second bush 28 or via the outer circumference of the tube 29. Together, the spring bundle 27 and tube 29 can form a torsion spring unit as part of the torsion spring assembly 20. The tensioning arm 19 is thus resiliently rotatable relative to the second bush 28 around the axis A_2. If suitably dimensioned, the sleeves 30, 31 can serve as bearing sleeves or friction damping sleeves if the upper end of the tube 29 is rotatable relative to the first bush 26. The sleeves 30, 31 can also be used as rotary stop sleeves for the upper end of the tube 29, if the tube 29 is to serve as a tube spring which is sequentially connectable to the torsion spring bundle 27 and which, from a certain angle of rotation of the torsion spring assembly onwards, is to increase the spring rate. The sleeves 30, 31 can also serve as clamping sleeves which produce a rotationally fast connection between the upper end of the tube 29 and the first bush 26, so that the tube is connected functionally in parallel to the torsion spring bundle 27, increasing the spring rate over the entire angle of rotation of the torsion spring assembly. As the spring bundle 27, when under torsion, is slightly shortened, the ends of the spring bundle can be held in a rotationally ~~fast~~ fixed way in at least one of the first and second bushes 26, 28 or in both, but they can be slightly axially displaceable.

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Please replace paragraph [0070] 78 with the following amended paragraph:

[0070] 78 Figure 6 shows the inventive belt tensioning device with its major components with the first tensioning roller 15, the first tensioning arm 19', the second tensioning roller 35, the second tensioning arm 39 and a torsion spring assembly 20. The rollers are advantageously made of lightweight plastic parts which are rotatably supported at the ends of their tensioning arms 19', 39. The first tensioning arm 19' is a straight carrier which is held in a rotationally ~~fast~~ fixed way at the upper end of the torsion spring assembly. The second tensioning arm 39 is a cranked beam which is held below the first one on the torsion spring assembly 20' in such a way that the tensioning rollers 15, 39 run in one plane. The torsion spring assembly 20' is preferably supported in such a way that it is freely rotatable in the engine block.

A.M. 1/2/07

Please replace paragraph [0072] 80 with the following amended paragraph:

[0072] 80 Figure 8 shows the complete assembly according to Figure 7B in a larger scale. The first tensioning arm 19' with the first tensioning roller 15 is shown in section in the region of the torsion spring assembly 20' only, whereas the second tensioning arm 39 with the second tensioning roller 35 is shown in section in the center. It can be seen that the pockets 42 provided for weight-reducing purposes are provided on one side of the first tensioning arm 19 although, again, they could be through-apertures. At the free end of the second tensioning arm 39, there is shown the bearing journal 41 on which the second tensioning roller 35 is rotatably supported by means of a roller bearing 43. The inner bearing race of the bearing 43 is held by a screw 44 on the bearing journal 41. The second tensioning roller 35 embraces the outer bearing race of the bearing 43 in a form-fitting and positive way. The second tensioning roller 35 is thus rotatable around an axis of rotation A_2. At the sectioned end of the first tensioning arm 19', there is provided an eye 25 into which there is inserted a first bush 26 in a rotationally ~~fast~~ fixed way. Into the first bush 26, there is inserted one end of a bundle 27 of torsion springs 32, which spring bundle 27, in its entirety, is connected to the first bush 26 in a rotationally ~~fast~~ fixed way The other end of the bundles 27 of torsion springs 32 is inserted into a second bush 28, which bundles 27, again, in its entirety, is connected to said second bush 28 in a rotationally ~~fast~~ fixed way. A tube 29 connected to the second bush 28 in a rotationally ~~fast~~ fixed way is placed onto the first and second bushes 26, 28. The above-described end of the tube 29 extends over the lower end of the fist bush 26, with sleeves 30, 31 being inserted between the tube 29 and the first bush 26. If dimensioned accordingly, the sleeves 30, 31 can serve as bearing sleeves or friction damping sleeves. The second tensioning arm 39 is placed in a rotationally ~~fast~~ fixed way onto the upper end of the tube 29. The second tensioning arm 39 is supported axially downwardly via a spacing sleeve 40 which is positioned on a flange 38. Between the tube 29 and the spacing sleeves 40, there are positioned centering sleeves 36, 37. The second tensioning arm 39 is supported upwardly via plate springs 33 and friction discs 34 on the first tensioning arm 19', which plastic springs 33 and friction discs 34, together, form a friction damping unit. In this way, the bundle 27 of

torsion springs 32 and the tube 29 form spring units which are connected functionally in series and which can be incorporated under a respective pretension relative to the belt. If, alternatively, the second bush 28 and the lower end of the tube 29 are firmly clamped into holding mechanism, the first tensioning arm 19' with the first tensioning roller 15 is supported via the bundle 27 on the holding mechanism and, independently thereof, the second tensioning arm 39 with the second tensioning roller 35 is supported via the tube spring 29 on the holding mechanism. As the spring bundle 27, when under torsion, is slightly shortened, the ends of the spring bundle 27 can be held in a rotationally ~~fast~~ fixed way in at least one of the first and second bushes 26, 28 or in both, but they can be slightly axially displaceable.

Please replace paragraph [00~~82~~]90 with the following amended paragraph: AM 1/2/07

[00~~82~~]90 Figure 15 shows in greater detail the design of the assembled tensioning arm 19 comprising the two arm halves 48, 49. A bearing journal 21 is inserted into the bearing aperture prior to the two halves 48, 49 being assembled. The tensioning roller 15 is rotatably supported in the tensioning arm 19 by way of a roller bearing 23 not shown in detail. At the opposite end, the sleeve 50, at its end face, is connected in a rotationally ~~fast~~ fixed way to a first bush 26 which is rotatably inserted into a tube 29 and forms a bearing 54 therewith. A torsion spring assembly 32, by means of its upper end, is inserted in a rotationally ~~fast~~ fixed way into the first bush 26. The lower end of the torsion spring 32 is inserted in a rotationally ~~fast~~ fixed way into a second bush 28. The tube 29 is axially connected in a rotationally ~~fast~~ fixed way to the second bush 28. Between the sleeve 50 and the upper end of the tube 29 there is shown a multi-component sleeve which can form a friction damping element 55 between the eye 25 and the upper end of the tube 29. In accordance with the invention, the central plane of the movement E of the tensioning roller 15 and of the tensioning arm 19, which, at the same time, forms the diving plane between the halves 48 and 49, is positioned within the axial region A of the bearing 54. More particularly, the plane E is positioned approximately centrally relative to the axial length A of the bearing 54. The bearing 54 between the tensioning arm 19 and the tube 29 is thus free from transverse forces and bending moments, so that optimum functioning of the bearing is ensured.

Please replace paragraph [00~~83~~]91 with the following amended paragraph: A.M. 1/2/07

[00~~83~~]91 Figure 16 shows an inventive belt tensioning device in a further embodiment with a first tensioning roller 15, a first tensioning arm 19', a second tensioning roller 35, a second tensioning arm 39', and a torsion spring assembly 20'. The first tensioning arm 19', which is connected in a rotationally ~~fast~~ fixed way to a bush 26 shaped cover-like, at its tope end, is divided yoke-like, whereas the second tensioning arm 39' which his connected to the tube 29 of the torsions spring assembly

20' engages between the two yoke parts. It can be seen that a torsion spring 32 form-fittingly engages the bush 26. The two tensioning arms 19' and 39' are each composed of two halves, as shown in greater detail at the tope end of Figure 14.

Please replace paragraph [008~~4~~]92 with the following amended paragraph:

AM. 1/2/07

[008~~4~~]92 Figure 17 shows the device according to Figure 16 in a partial section through the central axis A_2 of the torsion spring 32. The torsion spring 32, at its top end, is form-fittingly inserted into the first bush 26 which, via its cover-like shape, is connected to the first tensioning arm 10' in a rotationally ~~fast~~ fixed way and which, at its lower end, is connected to a second bush 28 in a rotationally ~~fast~~ fixed way. The second bush 28, in turn, is connected to the tube 29 in a rotationally ~~fast~~ fixed way. The second tensioning arm 39' which, as can be seen, also consist of two halves, is firmly placed onto the upper end of the tube 29. In this embodiment, too, the fist bush 26 and the tube 29 form a bearing region 54 with one another. Inside the eye 25 of the tensioning arm 19' there is positioned a further sleeve 55 which forms a friction damping element which is effective between the upper end of the tube 29 and the eye 25. In the present case, the tube 29 is surrounded by a spacing sleeve 40 relative to which the tube 29 is freely rotatable. The spacing sleeve 40 serves to secure the rotary spring assembly 20' in the rack. The central plane of movement E of the tensioning rollers and tensioning arms (not illustrated in this Figure), which, at the same time, forms the central dividing plane of the two tensioning arms 19' and 39', in this embodiment, too, in accordance with the invention is positioned centrally relative to the axial length A of the bearing 54 between the first bush 26 and the tube 29, so that the bearing assembly is free from transverse forces and bending moments and operates in an interference-free way, even if high forces are generated between the first and second tensioning arms 19', 39'.